





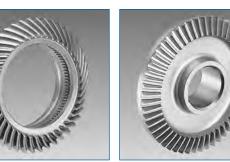
STOCK SPIRAL **BEVEL GEARS**

We stock 51 different sets of lapped spiral bevel gears in ratios of 1 to 1, 2 to 1, 3 to 1, 3 to 2 and 4 to 3 and 8 different sets of ground tooth spiral bevel gears in ratios of 1 to 1 and 2 to 1. Should you be unable to satisfy your gear requirements from the selection of stock gears listed in our catalog, please contact us for assistance. We can modify most of our stock gears to your specifications.

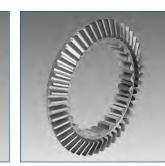
Custom Gears

Spiral Bevel

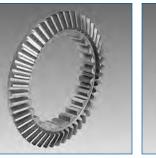
In addition to the stock gears listed in this catalog we manufacture spiral bevel, hypoid, Zerol® bevel, Coniflex® bevel, helical and spur gears, as well as Curvic® couplings to customers' prints and specifications. Please refer to the following chart for the complete range of sizes and capabilities.



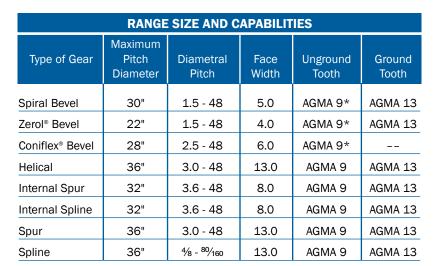




Coniflex® Bevel



Spur



^{*}Some Configurations to AGMA Quality Number 10 (lapped).



TYPES OF GEARS AND SIZES
RATING DATA AND
SPECIFICATIONS
STANDARD STOCK GEAR
SPECIFICATIONS
China Gear-STAN®
NON-STOCK GEARS
STOCK GROUND TOOTH GEARS12
DESIGN CONSIDERATIONS
Bearing Loads
Installation
Mountings
Bearing Pattern
Application



MARREAL

Helical



Curvic® Coupling



Standard and ground tooth stock spiral bevel gears ...

RATING DATA AND SPECIFICATIONS

China Gear stock gears are lapped to AGMA Quality Number 9 or ground to AGMA Quality Number 11. Each pair of gears is made of alloy steel with carburized and hardened teeth. 20° pressure angle and 35° spiral angle are standard. All pinions are left hand spiral. Mounting distance, backlash, mating teeth and set number are etched on each pair. See page 16.

Hub type gears can be rebored to the maximum diameter specified in the tables. It is preferred that all remachining of bores be performed by China Gear Manufacturing Inc.

HP = Horsepower

Calculations

 $T_W = HP \times 63025$

RPM	••	Working torque(in. lb.)Revolutions/minute
$T_r = T_W \frac{SF}{K_V}$	т _а т _r	= Allowable torque (in. lb.)= Catalog torque (in. lb.) (SF = 1)
	K _V	= Velocity Factor = $\sqrt{\frac{78}{78 + \sqrt{PLV}}}$ (Lapped AGMA Q9) = 1 (Ground AGMA Q11)
	PLV	Pitch line velocity0.262 x RPM x PitchDiameter

Service factors have been determined by many industries for specific applications from field data and should be used when available. In the absence of a service factor, select an appropriate overload factor.

= Service Factor

OVERLOAD FACTORS											
POWER CHARACTER OF LOAD SOURCE ON DRIVEN MACHINE											
Uniform Light Shock Medium Shock	Uniform 1.00 1.25 1.50	Medium Shock 1.25 1.50 1.75	Heavy Shock 1.75 2.00 2.25								

China Gear Stock Gear Selection

1) Calculate the pinion working torque (T_{WD}) .

$$T_{Wp} = \frac{63025 \times HP}{RPM_p}$$

 $\label{eq:RPMp} \text{RPM}_p$ 2) Estimate the rated pinion torque (T $_{rp}$).

$$T_{rp} = 2 \times T_{wp}$$

- 3) Find the rated pinion torque in the catalog that is approximately equal to the estimated torque.
- 4) Calculate the pitch line velocity (PLV).PLV = 0.262 x pinion pitch diameter x RPM_D
- 5) Calculate the dynamic factor K_V.

$$K_{V} = \sqrt{\frac{78}{78 + \sqrt{PLV}}}$$

6) Calculate the allowable pinion torque (Tap).

$$T_{ap} = T_{rp} \times K_{v}$$

7) Calculate the service factor.

Example

Customer requires a bevel 3:1 reduction

Pinion speed = 1800

$$HP = 38$$

Then:
$$T_{Wp} = \underline{63025 \times 38} = 1330 \text{ in. lb.}$$

1800

First estimate

$$T_{rp} = 2 \times 1330 \text{ in. lb.} = 2660 \text{ in. lb.}$$

From the 3:1 ratios on page 7

(6P45L15/6P15R45):

 T_{rp} = 2381 in. lb. (catalog value)

PLV = 0.262 x 2.5 x 1800 = 1179

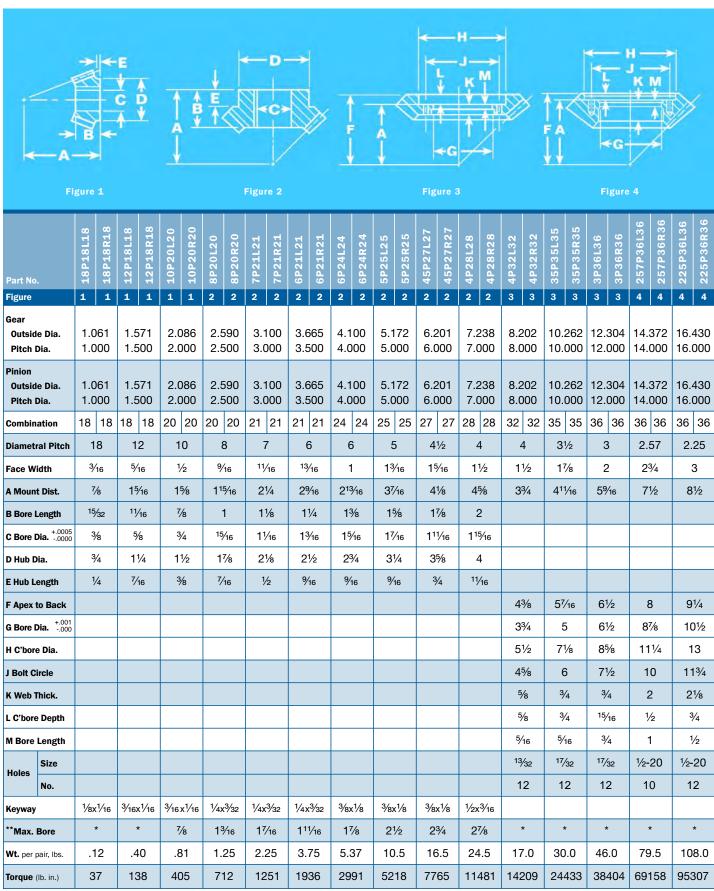
$$K_V = \sqrt{\frac{78}{78 + \sqrt{1179}}} = 0.833$$

$$T_{ap} = 2381$$
 in. lb. x 0.833 = 1983 in. lb.

A 1.49 SF indicates that the stock gear set has a capacity of 1.49 times that required.

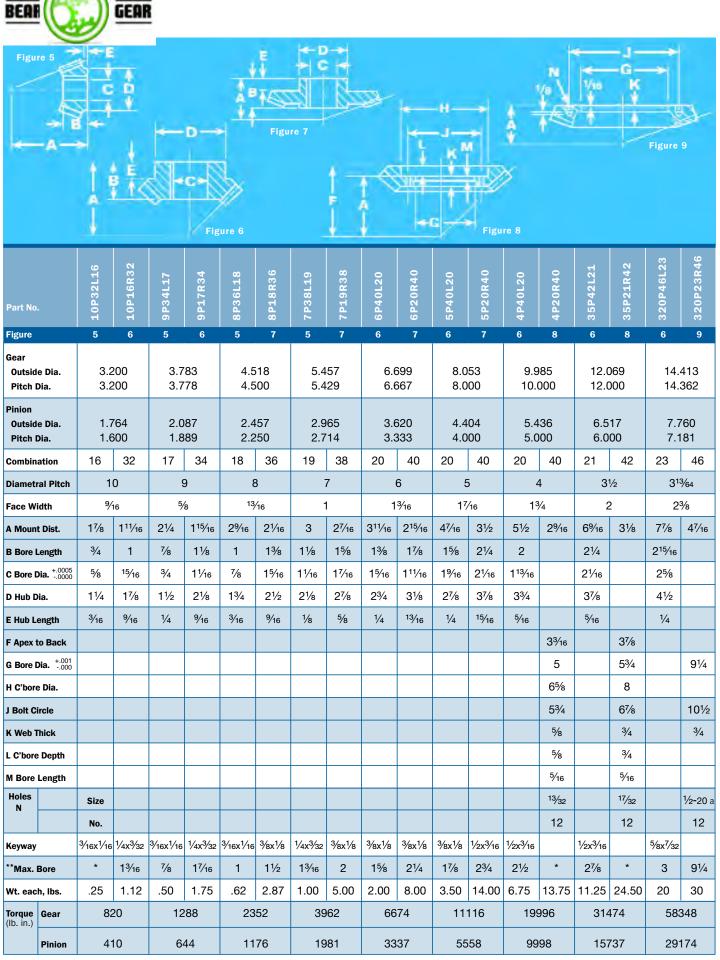
Gear sizes in this manual must be selected from the calculated allowable torque. For applications involving unusual conditions, our Engineering Service is available.





*Cannot be reworked.

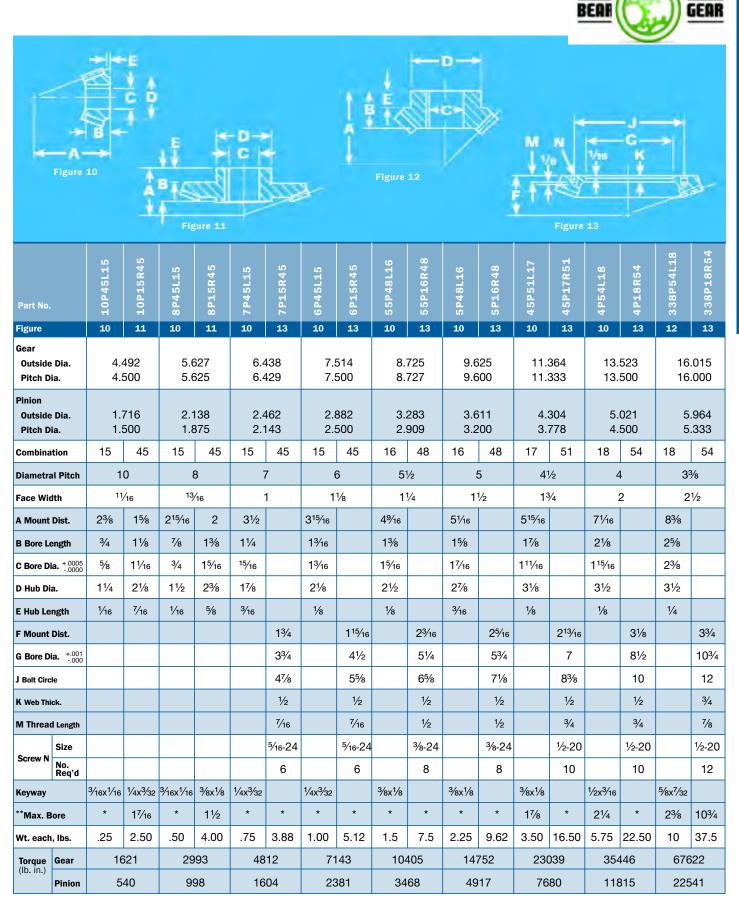
**Keyway Unchanged



*Cannot be reworked.

**Keyway Unchanged

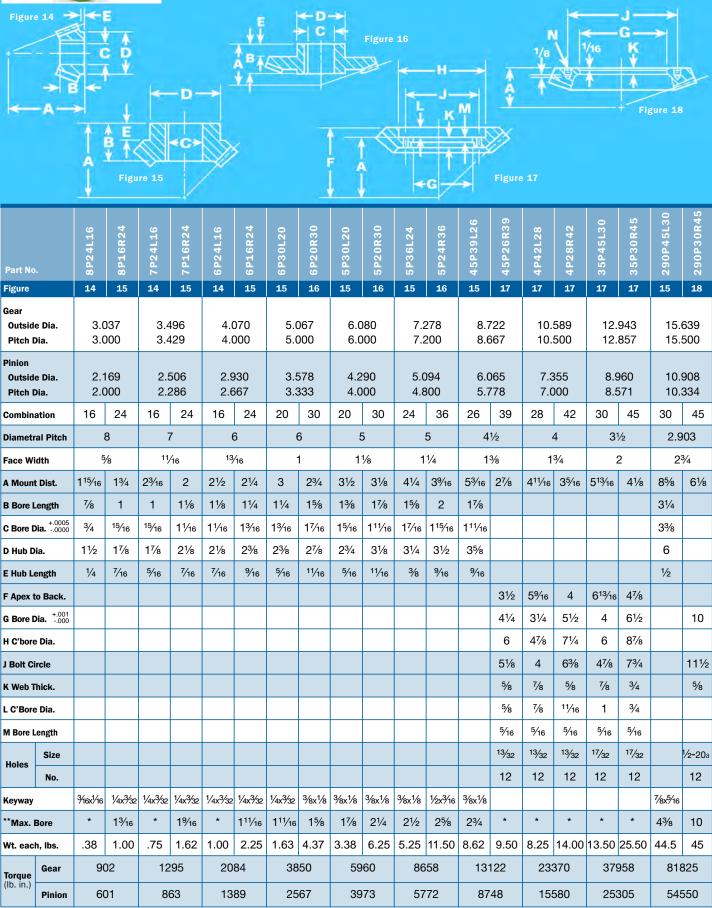
a - 3/4 thread length



*Cannot be reworked.

**Keyway Unchanged





*Cannot be reworked.

**Keyway Unchanged

a - 1/8 thread length





*Cannot be reworked.

^{**}Keyway Unchanged

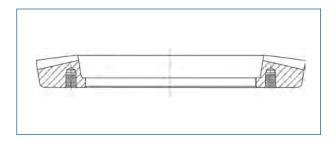


China Gear-STAN® Standard (Non-Stock) Ratios

The combinations listed in the following pages represent a line of Spiral Bevel Gears in sizes larger than our general selection of stock gears. We are tooled to produce these gear combinations without undue delays other than the normal time needed for the machining processes.

They are listed in groups according to the Pitch Diameter of the *gear*, with a suitable selection of ratios to cover a wide range of applications. (Please contact our Design Engineering Department for other sizes and ratios.)

All ring gears are carburized and *die quenched* on the most modern type of equipment available, and kept to the closest possible limits of flatness and roundness.



China Gear-Stan gear style used on ring gears.

As in our stock gear line, capacities are rated in terms of torque. The allowable torque, as shown on page 4, must be calculated before selecting gear size.

Ring gears should be ordered as shown in the following tables to take advantage of extensive tooling available. Pinion members can be designed to suit your machine or housing. Pinions of ratios higher than 3:1 are usually designed integral with the shaft because of fastening problems.

14 INCH PITCH DIAMETER OF GEAR

	SIZES	S		SPECIFICATION							DES		CAPACITY		
0.0).	Pitch	Pitch Dia. Ratio		Combi-	Diam.	Face		nting ance	Bore	Bolt Circle	No. of	Bolt Size	Torque Pinion	Torque Gear
Gear	Pinion	Gear	Pinion		nation	Pitch	Width	Gear	Pinion (Min.)		Dia.	Bolts	O.L.O	Lb. Inches	Lb. Inches
14.027	5.973	14	5.50	2.55	22-56	4.00	21/4	3½	73/4	9.250	10.500	12	1/2-20	16950	43145
14.030	5.220	14	4.75	2.95	20-59	4.21	21/4	31/4	7¾	9.250	10.500	12	1⁄2-20	14183	41840
14.019	3.993	14	3.55	3.94	17-67	4.79	21/4	2¾	7½	9.250	10.500	12	1⁄2-20	9839	38777
13.990	3.517	14	3.09	4.53	15-68	4.86	21/4	25/8	7½	9.250	10.500	12	1/2-20	8367	37930

16 INCH PITCH DIAMETER OF GEAR

16.040	6.837	16	6.28	2.55	22-56	3.50	2½	4	8½	10.750	12.000	12	1/2-20	23790	60556
15.950	4.599	16	4.06	3.94	17-67	4.19	2½	3	8½	10.750	12.000	12	1⁄2-20	13810	54428
16.019	3.720	16	3.24	4.93	15-74	4.63	2½	21//8	81/2	10.750	12.000	12	1⁄2-20	10380	51208



18 INCH PITCH DIAMETER OF GEAR

	SIZES	;			SP	ECIFIC	ATION				DESI	GN		CAPACITY		
0.1	D.	Pitch Dia.		Ratio	Combi-	Diam.	Face		nting ance	Bore	Bolt Circle	No. of	Bolt Size	Torque Pinion	Torque Gear	
Gear	Pinion	Gear	Pinion		nation	Pitch	Width	Gear	Pinion (Min.)		Dia.	Bolts	0.20	Lb. Inches	Lb. Inches	
18.196	18.196	18	18	1.00	39-39	2.17	3 ¹³ ⁄16	10	10	10.750	13.375	12	1/2-20	147373	147373	
18.062	7.980	18	7.33	2.46	22-54	3.00	2¾	4¾	9½	12.500	14.125	12	1/2-20	33827	83030	
18.015	6.558	18	5.89	3.06	18-55	3.06	2¾	4	9½	12.500	14.125	12	1/2-20	26219	80114	
18.013	5.235	18	4.58	3.93	15-59	3.28	23/4	3½	91⁄4	12.500	14.125	12	1⁄2-20	19457	76531	
18.034	4.235	18	3.65	4.93	14-69	3.83	2¾	3	91⁄4	12.500	14.125	12	1⁄2-20	14362	70784	

20 INCH PITCH DIAMETER OF GEAR

20.218	20.218	20	20	1.00	39-39	1.95	41/4	11	11	12.500	14.625	12	5⁄8 - 18	196902	196902
20.086	8.795	20	8	2.5	20-50	2.50	3	51⁄4	10½	13.875	15.500	12	%-18	44196	110490
20.025	7.987	20	7.47	2.68	28-75	3.75	3	43/4	10½	13.875	15.500	12	%-18	36101	96702
20.026	7.466	20	6.93	2.88	26-75	3.75	3	4½	10½	13.875	15.500	12	5 ⁄8 - 18	33291	96032
20.023	6.318	20	5.71	3.50	20-70	3.50	3	4	10½	13.875	15.500	12	5⁄8 - 18	27588	96558
20.015	5.641	20	5.07	3.95	19-75	3.75	3	3½	10½	13.875	15.500	12	%-18	23520	92842
20.012	4.856	20	4.27	4.69	16-75	3.75	3	31/4	10½	13.875	15.500	12	5 ⁄8-18	19418	91022

22 INCH PITCH DIAMETER OF GEAR

22.175	22.175	22	22	1.0	39-39	1.77	443/64	121⁄4	121⁄4	13.875	16.000	12	3⁄4-16	255914	255914
22.079	8.069	22	7.33	3.00	21-63	2.86	31⁄4	43⁄4	11½	15.000	17.000	12	¾- 1 6	43419	130257
22.042	6.171	22	5.50	4.00	18-72	3.27	31/4	4	11½	15.000	17.000	12	³ ⁄ ₄ -16	30092	120372
22.012	5.633	22	4.99	4.41	17-75	3.41	31/4	3¾	11½	15.000	17.000	12	³ ⁄ ₄ -16	26602	117362
22.010	5.057	22	4.40	5.00	15-75	3.41	31⁄4	3½	11½	15.000	17.000	12	3⁄4-16	23122	115610

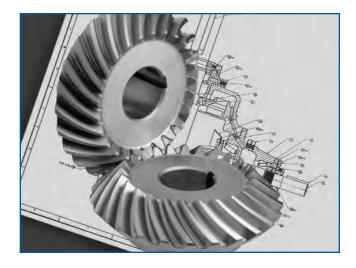
24 INCH PITCH DIAMETER OF GEAR

24.243	24.243	24	24	1.00	42-42	1.75	57/64	131⁄4	131⁄4	15.250	17.000	12	3⁄4-16	319016	319016
24.045	11.932	24	11.37	2.11	36-76	3.17	31/4	6¾	12½	17.500	19.500	12	3⁄4-16	69528	146781
24.041	9.667	24	9.00	2.67	27-72	3.00	31/4	5½	12½	17.500	19.500	12	¾-16	54907	146419
24.048	8.586	24	7.73	3.11	19-59	2.46	31/4	5	12½	17.500	19.500	12	3⁄4-16	49704	154344
24.027	8.146	24	7.44	3.23	22-71	2.96	31/4	5	12½	17.500	19.500	12	3⁄4-16	44722	144330
24.012	7.585	24	6.86	3.50	20-70	2.92	31⁄4	4½	12½	17.500	19.500	12	3⁄4-16	41120	143920
23.985	6.141	24	5.44	4.41	17-75	3.13	31⁄4	4	12½	17.500	19.500	12	3⁄4-16	30977	136667



Now...from the spiral bevel gear specialists

GROUND TOOTH SPIRAL BEVEL GEARS ...FROM STOCK



China Gear Manufacturing Inc. was the first gear manufacturer to offer ground tooth spiral bevel gears . . .

from stock. The most popular sizes of 1:1 and 2:1 ratios are currently available for time-saving, off-the-shelf delivery.

Every stock ground tooth gear is designed and manufactured to fulfill the following requirements for discriminating gear buyers.

Speeds in Excess of 8,000 SFPM

Ground tooth spiral bevel gears should be used for speeds exceeding 8000 surface feet per minute. Ground tooth spiral bevel gears make velocity factor devaluation unnecessary. (See page 4.) A constant velocity factor of 1.00 means you transmit more torque or horsepower . . . up to 30% more with the same size gear and pinion.

Reduce Gear Noise

Ground tooth spiral bevel gears are a design "must" at high speeds to reduce the decibel level of your gear box. Tooth contact ratios are maintained to a minimum of 2.0 to assure quiet operations.

Eliminate Positioning Errors

To achieve near "zero" positioning error, designers and manufacturers of radar systems, navigational gear, printing presses and machine tools specify ground tooth spiral bevel gears.

Higher Quality

All China Gear ground tooth spiral bevel gears are manufactured to AGMA Quality Number 11 or better.

High Capacity

Have your gear capacity requirements outgrown your present housing and mountings? Eliminate unnecessary redesigning or gear box size increases. Investigate the possible use of ground tooth spiral bevel gears for increased capacity. All China Gear ground tooth gears are shot peened for additional fatigue life.

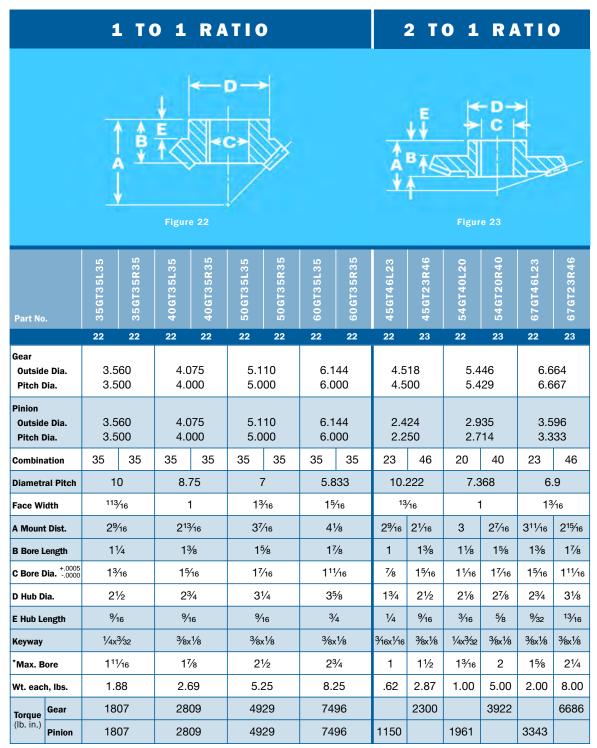
Uniform Load-Carrying Capabilities

Grinding gear teeth corrects heat treat distortion to minimize tooth spacing errors and increase load capacity.

Arrow's On Demand Program for Ground Tooth Spiral Bevel Gears

China Gear is able to produce ground tooth spiral bevel gears from a wide variety of our stock gears, and do so in a fraction of the time when compared to producing a ground tooth gear from scratch. This ability promises to offer many benefits to manufacturers of power transmission systems.





*Keyway Unchanged.



TANGENTIAL LOAD COEFFICIENTS FOR BEARING LOADS

The normal load on spiral bevel gear tooth surfaces may be resolved into three (3) components: (Wt) tangential; (Wx) axial and (Wr) radial.

The tangential and radial components act in a plane perpendicular to the gear axis and produce radial bearing loads. The axial component acts in a direction parallel to the axis producing thrust plus additional radial bearing loads.

The value of the axial and radial loads can be determined by multiplying the tangential load at mid face (Wtm) by the applicable coefficient (Kx) or (Kr) for the concave or convex load face of either the pinion (p) or the gear (g).

Fig. 24 is a table of coefficients (Kx) and (Kr) vs. gear ratios for 35° spiral bevel gears with 90° shaft angles and 20° pressure angle. Note the (+) values indicate forces tending to separate the two gears and the (-) values indicate forces drawing the gears into tighter mesh.

Fig. 24 - Tangential Load Coefficients for Bearing Loads

Coefficients for Spiral Bevel Gears: $\Sigma\text{=}90\,^{\circ}$ Shaft Angle/ $\phi\text{=}20\,^{\circ}$ Pressure Angle/ $\psi\text{=}35\,^{\circ}$ Spiral Angle

Load Face	Concave Pinion	Convex Gear	Convex Pinion	Concave Gear
Ratio Ng/np	Kxp=(Krg)	Kxg=(Krp)	Kxp=(Krg)	Kxg=(Krp)
1.0	.809	181	181	.809
1.1	.817	142	219	.800
1.2	.822	107	253	.790
1.3	.826	075	284	.779
1.4	.828	045	312	.769
1.5	.829	019	336	.758
1.6	.829	.006	358	.748
1.7	.829	.028	378	.738
1.8	.828	.048	396	.728
1.9	.827	.067	413	.719
2.0	.825	.084	428	.711
2.5	.815	.152	485	.673
3.0	.805	.200	524	.643
3.5	.795	.235	551	.620
4.0	.787	.261	572	.601
4.5	.780	.282	587	.586
5.0	.774	.298	599	.573
5.5	.768	.312	609	.562
6.0	.764	.323	618	.553
6.5	.760	.333	625	.546
7.0	.756	.341	630	.539
7.5	.753	.348	635	.533
8.0	.750	.354	640	.528
8.5	.747	.359	643	.523
9.0	.745	.364	647	.519
9.5	.743	.369	650	.515
10.0	.741	.372	653	.512

W_t = Tangential Load = 126050 HP d RPMp W_{tm} = Tangential Load W_t at Mid-Face = $d\sqrt{1+mg^2}$

2 \(\begin{aligned}
2 \left(1+mg)^2
\end{aligned}

 W_{y} = Axial Load Component = $K_{x}W_{tm}$ W_r = Radial Load Component = $K_r W_{tm}$

HP = Horsepower

d = Pinion Pitch Diameter

F = Face Width

mg = Ratio NG/NP

 $K_X = Axial Coefficient$

 $K_r = Radial Coefficient$

 $RPM_p = Pinion RPM$

rmp = pinion mean pitch radius rmg = gear mean pitch radius rmg = rmp x mg

Load Components Concave L.H.

Fig. 25 - Normal Tooth

Fig. 26 - Normal Backlash at **Tightest Point of Mesh**

Diametral Pitch	Backlash
1	.020" to .030"
2	.012" to .016"
3	.008" to .011"
4	.006" to .008"
6	.004" to .006"
10	.002" to .004"
20 and Finer	.001" to .003"

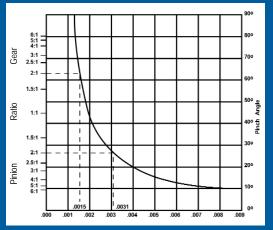


Fig. 27 - Axial Movement Per .001" Change in Backlash (Inches)



A means of inspecting the gears in mesh is desirable both from an assembly standpoint and for periodic check. An inspection hole and cover should be arranged so that the contact pattern can be observed on the teeth of both members of the gear set.

In storage or during shipment lapped gears should always be fastened together in pairs or sets, and they should not be separated until ready to assemble.

INSTALLATION

Mounting Distance

The correct setting or adjustment of the pinion at assembly is most important. Provision should be made for adjusting both the gear and pinion axially. It is advisable to first adjust the pinion to its correct mounting distance (See figure 28), determined by measurement or by a gage centered on the gear shaft or a "dummy" shaft made for this purpose. The gage may be arranged to measure from the center of the gear shaft to a flat on the extreme small end of the pinion teeth or to the back face of the pinion hub. After the pinion has been correctly positioned, the gear should then be adjusted to mesh with the pinion to obtain the desired amount of backlash.

The shims used in adjusting the gear and pinion location, and the bearing preload, should not be less than 0.015" thick and should preferably be on the stationary member of the bearing.

Backlash

Bevel gears should be manufactured and assembled to have a definite amount of backlash, which varies according to pitch and operating conditions. Backlash is necessary for safe operation. If gears are set too tight they will be noisy, wear excessively, and possibly scuff the tooth surfaces, or even break. Figure 27 shows the ratio at which the axial movement of either member affects the backlash.

Figure 26 suggests the recommended normal backlash at tightest point of mesh for gears assembled, ready to run. The backlash values etched on China gears are derived from this table and apply to the tightest point of mesh. (See also Figure 29). In many instances, these limits will require modifications to suit the special conditions of operation.

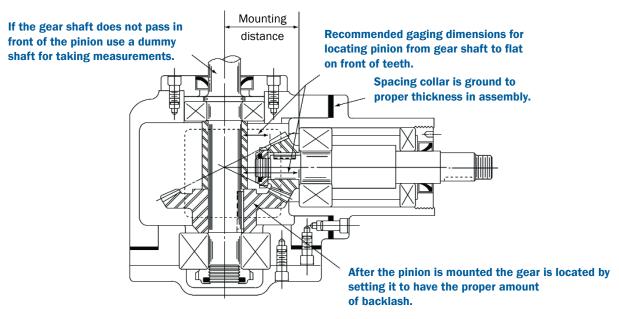


Fig. 28 - Measuring or gaging as shown is the recommended method for locating the pinion. Pinion should be set to mounting distance marked on pinion, and gear should be adjusted to give correct backlash.

Courtesy: The Gleason Works



MOUNTINGS

Rigid mountings should be provided to hold the displacements of the gears under operating loads within recommended limits. Care should be taken to see that keys are hardened, properly fitted and that couplings are not out of true or out of square.

For a number of years the Gleason Works has been making deflection tests on gears and their mountings and observing these same units in service. From these tests the recommended allowable deflections under maximum service load have been determined for gears from 6" to 15" diameter:

- 1. The pinion should not lift or depress more than 0.003".
- 2. The pinion should not yield axially more than 0.003" in either direction.
- The gear should not lift or depress more than .003".
- 4. The gear should not yield axially more than 0.003" in either direction on miters or near miters or more than 0.010" away from the pinion on higher ratios.

Spiral bevel gears should in general be mounted on anti-friction bearings in an oil-tight case. While designs may be made for a given set of conditions using plain bearings for radial and thrust loads, the problem of maintaining the gears in satisfactory alignment is usually more easily accomplished with ball or roller bearings.

There are two general types of pinion mountings, namely the straddle and the overhung mounting. Either ball or roller bearings may be used in both types of mountings.

Ball bearings with extremely small axial yield should be used behind each pinion to take care of combined thrust and radial loads. Matched angular contact or double row deep groove angular contact bearings are preferred. At the other end of the shaft a single row radial bearing may be used as shown in Figures 30 and 33.

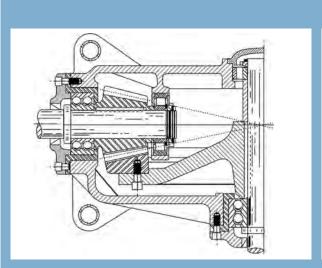
When mounted on taper roller bearings, the indirect mounting should be used. That is, the large ends of the tapered rollers of each bearing should point outward as shown in Figures 31 and 32. The thrust load of the pinion is thus absorbed by the bearing adjacent to the pinion and the reverse thrust load will be taken by the opposite bearing.

In either type of mounting both the gears and thrust bearings should be locked against thrust in either direction. This applies to straight bevel and Zerol® bevel gears as well as to spiral bevel and hypoid gears. It is accepted practice to preload the bearings to remove initial freedom in the mounting. The amount of preload depends upon the mounting load and operating speed, and should be established by the bearing manufacturer.



Fig. 29 - All China Gear Stock Gears are marked with the above assembly information.





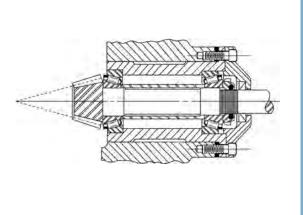
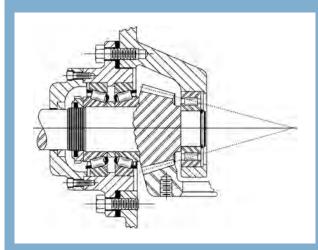


Fig. 30 - Typical straddle mounting for both members of a spiral bevel pair

Fig. 31 - This mounting is another form of bearing arrangement for overhung pinions.



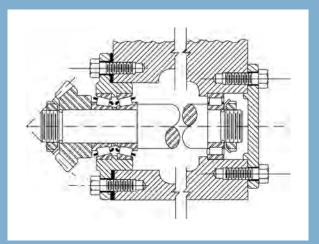


Fig. 32 - Straddle pinion mounting for short shafts showing use of combined thrust and radial bearings. Gear mounted in oil-tight case.

Fig. 33 - Arrangement recommended for long shafts to prevent temperature changes affecting position of gear mounted in oil-tight case.

Acknowledgment is gratefully extended to Gleason Works, Rochester, New York and to the American Gear Manufacturers Association for portions of the text and illustrative material used in this section.



BEARING PATTERN

Using a suitable marking compound, check the bearing pattern. If the markings on the gear set have been followed, the pattern will conform to accepted standards.

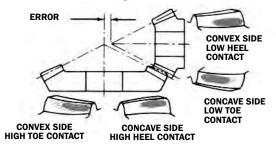
Gears are cut with a contact pattern about half the length of the tooth, the location slightly



favoring the toe end of the tooth. Under load the pattern will shift somewhat toward the heel of the tooth, and will thus become more central. Under no circumstances must the pattern be concentrated on the ends of the teeth.

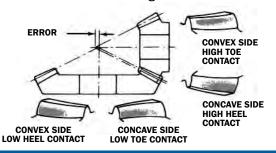
PROFILE ERROR

To correct: decrease mounting distance



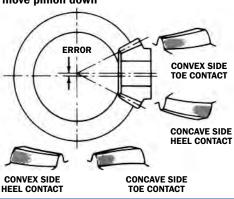
PROFILE ERROR

To correct: increase mounting distance



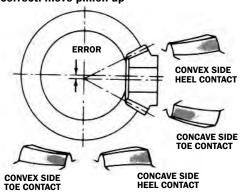
CROSS CONTACT

To correct: move pinion down



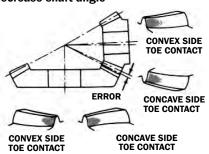
CROSS CONTACT

To correct: move pinion up



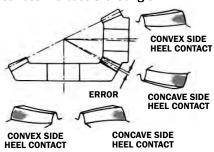
SHAFT ANGLE ERROR

To correct: decrease shaft angle



SHAFT ANGLE ERROR

To correct: increase shaft angle



(Note: Pinion member is left hand in all illustrations.)

All Illustrations: Courtesy of The Gleason Works.

APPLICATION ENGINEERING INFORMATION



GEARS AND GEARDRIVES

	State _			
	State _			
	State _			
		ZiP	Cou	ntry
	rax			
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e	Producti	on		
	Exact		ducer 🗖	Increaser \square
Uniform \square	Med. Shock \square	Hi Shock \square		
me Mover				
Pitch No of teeth Pr Angle Spiral Angle Shaft Angle AGMA Class Material AT YOU SEND AN A and direction of robing axial & radial justing backlash ecting the gearset	B10 Ove Typ SSEMBLY PRINT OF tation gear loads	ft Requirements: Parallel Angle Other y Cycle D Life rhung load e of Lub	□ Inters	sect Skew hrs
L DESIGN PARAME	ETERS			
	E: Yes No Input HP No Input H	Exact	g: Yes No	Exact Reducer



China Gear Manufacturing Inc.

No. 49, Salkang Avenue, Furong Industrial Zone

Shajing, Baoan, Shenzhen, China 518125

inquiry.gear@made-parts.com

Phone: +86-755-81498860

Fax: +86-755-81498829

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